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Numerical Simulation of Elongated Fibres in Horizontal Channel Flow

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Introduction

- In engineering predictions of dispersed two-phase flow spherical shape for particles is the general assumption.
- Most practical situations particles are irregular or have certain shape (e.g., granulates or fibres)
- Dynamics of non-spherical particles are substantially different from that of spherical (i.e., pitching and rotational torques)
- Analytical expressions of forces & moments are known for non-spherical regular particles in Stokes regime (Re_p << 1)
- For moderate Re_p, not much information available
- In such case, fully resolved DNS can be used to extract flow coefficients depending on Re_p and particle shape (e.g., Hölzer & Sommerfeld, 2009; Vakil & Green, 2009; Zastawny et al., 2012)
- Final goal: develop an engineering tool to predict dispersed two-phase flows laden with non-spherical particles

Numerical approximation

- System of interest: behaviour of non-spherical particles immersed in turbulent channel flow
- System described by the Euler-Lagrange approach
- Particles considered as point masses with dynamics given by linear and angular (orientation) momentum equations
- Fluid field computed by RANS (Reynolds Stress Model), modified by the presence of particles (two-way coupling)
- Non-spherical particles motion due to drag and lift forces, whose coefficients were previously obtained by DNS, as well as pitching and rotational torques.
- Here, elongated fibres at intermediate Reynolds numbers are considered. Expressions for the interaction with solid walls have been developed.
- Outputs: particle mean velocity, fluctuating velocity components (streamwise and vertical) and concentration profiles in the channel.

Non-spherical particles governing equations



$$\vec{x'} = A \cdot \vec{x''}$$

Rotation matrix **A** written in terms of Euler parameters

$$\mathbf{A} = \begin{bmatrix} 1 - 2\left(\varepsilon_{2}^{2} + \varepsilon_{3}^{2}\right) & 2\left(\varepsilon_{1}\varepsilon_{2} + \varepsilon_{3}\eta\right) & 2\left(\varepsilon_{1}\varepsilon_{3} - \varepsilon_{2}\eta\right) \\ 2\left(\varepsilon_{1}\varepsilon_{2} - \varepsilon_{3}\eta\right) & 1 - 2\left(\varepsilon_{1}^{2} + \varepsilon_{3}^{2}\right) & 2\left(\varepsilon_{3}\varepsilon_{2} + \varepsilon_{1}\eta\right) \\ 2\left(\varepsilon_{1}\varepsilon_{3} + \varepsilon_{2}\eta\right) & 2\left(\varepsilon_{3}\varepsilon_{2} - \varepsilon_{1}\eta\right) & 1 - 2\left(\varepsilon_{2}^{2} + \varepsilon_{1}^{2}\right) \end{bmatrix}$$

Time evolution of Euler parameters

$$\begin{bmatrix} \frac{d\varepsilon_1}{dt} \\ \frac{d\varepsilon_2}{dt} \\ \frac{d\varepsilon_3}{dt} \\ \frac{d\eta}{dt} \end{bmatrix} = \frac{1}{2} \begin{bmatrix} \eta \omega_{x'} - \varepsilon_3 \omega_{y'} + \varepsilon_2 \omega_{z'} \\ \varepsilon_3 \omega_{x'} + \eta \omega_{y'} - \varepsilon_1 \omega_{z'} \\ -\varepsilon_2 \omega_{x'} + \varepsilon_1 \omega_{y'} + \eta \omega_{z'} \\ -\varepsilon_1 \omega_{x'} - \varepsilon_2 \omega_{y'} - \varepsilon_3 \omega_{z'} \end{bmatrix}$$

Non-spherical particles governing equations



Forces: drag and lift

Torques: pitching and rotational

Flow coefficients cylindrical fibres

Cylinders Vakil & Green, C&F (2009)



Correlations flow coefficients obtained by DNS depending on particle Reynolds numbers, orientation and <u>aspect ratio</u> (AR)

$$C_{D,cyl} = \frac{F_D}{\frac{1}{2}\rho\tilde{u}^2 LD} \qquad \qquad C_{L,cyl} = \frac{F_L}{\frac{1}{2}\rho\tilde{u}^2 LD}$$

Torque coefficients follow the approach of Yin et al., CES (2003)



- β , γ and r_c depend on shape and orientation of particle
- β , γ and r_c are determined using real values of θ , ϕ and ψ before wall collision and are determined using an analytic model
- A orientation matrix A is necessary defined to get the orientation the particle before wall collision

Hard particle approximation



- I. Particle stops sliding in compression period
- II. Particle stops sliding in recovery period
- III. Particle slides throughout compression and recovery period

Additional Assumptions:

Collision is modeled by the coefficient of restitution and friction is modeled by Coulomb's law

Case I

Linear velocities

$$u^{(2)} = \omega_y^{(2)} r_c \cos \beta - \omega_z^{(2)} r_c \sin \beta \sin \gamma$$

$$v^{(2)} = \omega_z^{(2)} r_c \sin \beta \cos \gamma - \omega_x^{(2)} r_c \cos \beta$$

$$w^{(2)} = w^{(0)} + (1 + e) J_z^{(1)} / m_p$$
Angular velocities

$$\omega_x^{(2)} = \frac{l_x \omega_x^{(0)} - m_p v^{(0)} r_c \cos \beta - (1 + e) r_c \sin \beta \sin \gamma J_z^{(1)} + m_p \omega_z^{(2)} r_c^2 \sin \beta \cos \beta \cos \gamma}{l_x + m_p r_c^2 \cos^2 \beta}$$

$$\omega_y^{(2)} = \frac{l_y \omega_y^{(0)} + m_p u^{(0)} r_c \cos \beta + (1 + e) r_c \sin \beta \cos \gamma J_z^{(1)} + m_p \omega_z^{(2)} r_c^2 \sin \beta \cos \beta \sin \gamma}{l_y + m_p r_c^2 \cos^2 \beta}$$

$$\omega_z^{(2)} = \frac{l_z \omega_z^{(0)} + m_p v^{(0)} r_c \sin \beta \cos \gamma - m_p u^{(0)} r_c \sin \beta \sin \gamma + m_p \omega_y^{(2)} r_c^2 \sin \beta \cos \beta \sin \gamma - m_p \omega_x^{(2)} r_c^2 \sin \beta \cos \beta \cos \gamma}{l_z + m_p r_c^2 \sin^2 \beta}$$

Linear impulse normal to the wall

 $J_z^{(1)} = m_p(\omega_x^{(1)}r_c\sin\beta\sin\gamma - \omega_y^{(1)}r_c\sin\beta\cos\gamma - w_0)$

$$\omega_{y}^{(1)} = \frac{m_{p}l_{y}l_{22}\omega_{y}^{(0)} + m_{p}l_{x}l_{12}\omega_{x}^{(0)} + m_{p}l_{z}l_{31}\omega_{z}^{(0)} + m_{p}r_{c}\cos\beta u_{0}l_{1}^{2} - m_{p}r_{c}\sin\beta\cos\gamma v_{0}l_{2}^{2} - m_{p}r_{c}\sin\beta\cos\gamma w_{0}l_{3}^{2}}{m_{p}l_{y}r_{c}^{2}\sin^{2}\beta\sin^{2}\gamma(l_{x}+l_{z}+m_{p}r_{c}^{2}) + m_{p}l_{x}r_{c}^{2}\cos^{2}\beta\sin^{2}\gamma(l_{y}+l_{z}+m_{p}r_{c}^{2}) + m_{p}l_{y}r_{c}^{2}\cos^{2}\beta(l_{x}+l_{y}+m_{p}r_{c}^{2}) + l_{x}l_{y}l_{z}}}$$
$$\omega_{x}^{(1)} = \frac{m_{p}l_{x}l_{11}\omega_{x}^{(0)} + m_{p}l_{y}l_{23}\omega_{y}^{(0)} + m_{p}l_{z}l_{32}\omega_{z}^{(0)} + m_{p}r_{c}\cos\beta u_{0}l_{4}^{2} - m_{p}r_{c}\sin\beta\cos\gamma v_{0}l_{5}^{2} + m_{p}r_{c}\sin\beta\cos\gamma w_{0}l_{6}^{2}}{m_{p}l_{y}r_{c}^{2}\sin^{2}\beta\sin^{2}\gamma(l_{x}+l_{z}+m_{p}r_{c}^{2}) + m_{p}l_{x}r_{c}^{2}\cos^{2}\beta\sin^{2}\gamma(l_{y}+l_{z}+m_{p}r_{c}^{2}) + m_{p}l_{y}r_{c}^{2}\cos^{2}\beta(l_{x}+l_{y}+m_{p}r_{c}^{2}) + l_{x}l_{y}l_{z}}}$$

Case I

Extra relations

$$\begin{split} l_1^2 &= m_p r_c^2 \sin^2 \beta \cos^2 \gamma l_x + m_p r_c^2 \sin^2 \beta \sin^2 \gamma l_z + m_p r_c^2 \cos^2 \beta l_z + l_x l_z \\ l_2^2 &= m_p r_c^2 \sin \beta \sin \gamma \cos \beta l_z - m_p r_c^2 \sin \beta \sin \gamma \cos \beta l_x \\ l_3^2 &= m_p r_c^2 \sin^2 \beta \cos^2 \gamma l_x + m_p r_c^2 \sin^2 \beta \sin^2 \gamma l_x + m_p r_c^2 \cos^2 \beta l_z + l_y l_z \\ l_4^2 &= m_p r_c^2 \sin \beta \cos \gamma \cos \beta l_z - m_p r_c^2 \sin \beta \cos \gamma \cos \beta l_x \\ l_5^2 &= m_p r_c^2 \sin^2 \beta \cos^2 \gamma l_z + m_p r_c^2 \sin^2 \beta \sin^2 \gamma l_y + m_p r_c^2 \cos^2 \beta l_z + l_y l_z \\ l_6^2 &= m_p r_c^2 \sin^2 \beta \cos^2 \gamma l_y + m_p r_c^2 \sin^2 \beta \sin^2 \gamma l_y + m_p r_c^2 \cos^2 \beta l_z + l_y l_z \\ l_{11} &= m_p r_c^4 \sin^2 \beta \cos^2 \gamma + r_c^2 \cos^2 \beta l_z + r_c^2 \sin^2 \beta \sin^2 \gamma l_y + r_c^2 \sin^2 \beta \cos^2 \gamma (l_y + l_z) + l_y l_z / m_p \\ l_{22} &= m_p r_c^4 \sin^2 \beta \sin^2 \gamma + r_c^2 \cos^2 \beta l_z + r_c^2 \sin^2 \beta \cos^2 \gamma l_x + r_c^2 \sin^2 \beta \sin^2 \gamma \cos^2 \gamma (l_x + l_z) + l_x l_z / m_p \\ l_{12} &= l_{23} &= m_p r_c^2 \sin^2 \beta \sin \gamma \cos \gamma + r_c^2 \sin^2 \beta \sin \gamma \cos \gamma l_z \\ l_{31} &= m_p r_c^2 \sin \beta \sin \gamma \cos \beta + r_c^2 \sin \beta \sin \gamma \cos \beta l_x \\ l_{32} &= m_p r_c^2 \sin \beta \cos \gamma \cos \beta + r_c^2 \sin \beta \cos \gamma \cos \beta l_x \end{split}$$

Case II

Linear velocities

$$u^{(2)} = \omega_y^{(2)} r_c \cos\beta - \omega_z^{(2)} r_c \sin\beta \sin\gamma$$
$$v^{(2)} = \omega_z^{(2)} r_c \sin\beta \cos\gamma - \omega_x^{(2)} r_c \cos\beta$$
$$w^{(2)} = w^{(0)} + (1+e) J_z^{(1)} / m_p$$

Angular velocities

$$\omega_{x}^{(2)} = \frac{I_{x}\omega_{x}^{(0)} - m_{p}v^{(0)}r_{c}\cos\beta - (1+e)r_{c}\sin\beta\sin\gamma J_{z}^{(1)} + m_{p}\omega_{z}^{(2)}r_{c}^{2}\sin\beta\cos\beta\cos\gamma}{I_{x} + m_{p}r_{c}^{2}\cos^{2}\beta}$$
$$\omega_{y}^{(2)} = \frac{I_{y}\omega_{y}^{(0)} + m_{p}u^{(0)}r_{c}\cos\beta + (1+e)r_{c}\sin\beta\cos\gamma J_{z}^{(1)} + m_{p}\omega_{z}^{(2)}r_{c}^{2}\sin\beta\cos\beta\sin\gamma}{I_{y} + m_{p}r_{c}^{2}\cos^{2}\beta}$$
$$\omega_{z}^{(2)} = \frac{I_{z}\omega_{z}^{(0)} + m_{p}v^{(0)}r_{c}\sin\beta\cos\gamma - m_{p}u^{(0)}r_{c}\sin\beta\sin\gamma + m_{p}\omega_{y}^{(2)}r_{c}^{2}\sin\beta\cos\beta\sin\gamma - m_{p}\omega_{x}^{(2)}r_{c}^{2}\sin\beta\cos\beta\cos\gamma}{I_{z} + m_{p}r_{c}^{2}\sin^{2}\beta}$$

Linear impulse normal to the wall

$$J_{z}^{(1)} = \frac{m_{p}I_{x}I_{y}(\omega_{x}^{(0)}r_{c}\sin\beta\sin\gamma - \omega_{y}^{(0)}r_{c}\sin\beta\cos\gamma - w^{(0)})}{I_{x}I_{y} + m_{p}r_{c}^{2}[I_{x}(\varepsilon_{x}\mu\sin\beta\cos\beta\cos\gamma + \sin^{2}\beta\cos^{2}\gamma) + I_{y}(\varepsilon_{y}\mu\sin\beta\cos\beta\sin\gamma + \sin^{2}\beta\sin^{2}\gamma)]}$$

Case III

Linear velocities

$$u^{(2)} = u^{(0)} - \varepsilon_x \mu (1+e) \frac{J_z^{(1)}}{m_p}$$
$$v^{(2)} = v^{(0)} - \varepsilon_y \mu (1+e) \frac{J_z^{(1)}}{m_p}$$
$$w^{(2)} = w^{(0)} + (1+e) \frac{J_z^{(1)}}{m_p}$$

Angular velocities

$$\omega_x^{(2)} = \omega_x^{(0)} - (1+e)[\varepsilon_y \mu \cos\beta + \sin\beta \sin\gamma] \frac{J_z^{(1)} r_c}{l_x}$$
$$\omega_y^{(2)} = \omega_y^{(0)} + (1+e)[\varepsilon_x \mu \cos\beta + \sin\beta \cos\gamma] \frac{J_z^{(1)} r_c}{l_y}$$
$$\omega_z^{(2)} = \omega_z^{(0)} + (1+e)[\varepsilon_y \mu \cos\gamma - \varepsilon_x \mu \sin\gamma] \sin\beta \frac{J_z^{(1)} r_c}{l_z}$$

Linear impulse normal to the wall

$$J_{z}^{(1)} = \frac{m_{p}I_{x}I_{y}(\omega_{x}^{(0)}r_{c}\sin\beta\sin\gamma - \omega_{y}^{(0)}r_{c}\sin\beta\cos\gamma - w^{(0)})}{I_{x}I_{y} + m_{p}r_{c}^{2}[I_{x}(\varepsilon_{x}\mu\sin\beta\cos\beta\cos\gamma + \sin^{2}\beta\cos^{2}\gamma) + I_{y}(\varepsilon_{y}\mu\sin\beta\cos\beta\sin\gamma + \sin^{2}\beta\sin^{2}\gamma)]}$$

Conditions for occurrence of the cases

Case I & II

$$\frac{w^{(0)}}{u^{(0)} - \omega_y^{(0)} r_c \cos\beta + \omega_z^{(0)} r_c \sin\beta \sin\gamma} < -\frac{1}{(1 + m_p r_c^2 \cos^2\beta/l_y)\mu(e+1)}$$

Case III
$$\frac{1}{(1 + m_p r_c^2 \cos^2\beta/l_y)\mu(e+1)} < \frac{w^{(0)}}{u^{(0)} - \omega_y^{(0)} r_c \cos\beta + \omega_z^{(0)} r_c \sin\beta \sin\gamma} < 0$$

Above equations reduce in the 2D case to those of Sommerfeld (2002) and in the case of spherical particles to those of Crowe, Sommerfeld, Tsuji (1998).

Case IV, ideal collision ($e = 1, \mu = 0$)

Linear velocities

$$u^{(2)} = u^{(0)}$$
$$v^{(2)} = v^{(0)}$$
$$w^{(2)} = w^{(0)} + \frac{2J_z^{(1)}}{m_p}$$

(0)

(2)

Angular velocities

$$\begin{split} \omega_{x}^{(2)} &= \omega_{x}^{(0)} - 2\sin\beta\sin\gamma\frac{J_{z}^{(1)}r_{c}}{I_{x}}\\ \omega_{y}^{(2)} &= \omega_{y}^{(0)} + 2\sin\beta\cos\gamma\frac{J_{z}^{(1)}r_{c}}{I_{y}}\\ \omega_{z}^{(2)} &= \omega_{z}^{(0)} \end{split}$$

Linear impulse normal to the wall

$$J_{z}^{(1)} = \frac{m_{p} I_{x} I_{y} (\omega_{x}^{(0)} r_{c} \sin \beta \sin \gamma - \omega_{y}^{(0)} r_{c} \sin \beta \cos \gamma - w^{(0)})}{I_{x} I_{y} + m_{p} r_{c}^{2} \sin^{2} \beta (I_{x} \cos^{2} \gamma + I_{y} \sin^{2} \gamma)}$$

Simplified 2D equation cylinders (ideal case)



Fibre - wall interaction visualisation



Experimental rig

Channel Flow Configuration

Experimental studies by Kussin (2004)

Horizontal Channel: Length: 6 m Width: 350 mm Height: 35 mm $U_{av} = 20 \text{ m/s}$ Measuring section: 5.8 m



Spherical glass beads, $\rho_p = 2450 \text{ kg/m}^3$



Numerical set-up



Horizontal Channel:

6 m x 350 mm x 35 mm $U_{av} = 20 \text{ m/s}$ Measuring section: 5.8 m

 $\rho = 1.25 \text{ kg/m}^3$ $\mu = 1.8 \cdot 10^{-5} \text{ N} \cdot \text{s/m}^2$ $\rho_{\rm p} = 2450 \text{ kg/m}^3$ $D_p = 130 \ \mu m$ η = 0.1

Tracked particles

Spheres Fibres (aspect ratio 13)

Degree of roughness	Mean roughness in stream-wise direction [µm]	Mean roughness in lateral direction [µm]
R0 (LR)	2.32	2.09
R2 (HR)	6.83	6.89









Conclusions

- The Euler/Lagrange approach has been used to investigate the transport of spheres and elongated fibres in a turbulent channel flow
- Equations for the interaction of general non-spherical particles with walls have been presented in the context of "hard particles"
- Depending of angle of impact and rotational velocity one or two collisions with the wall can be observed.
- As expected, fibres follow better than spheres the fluid flow, as they tend to be aligned perpendicular to the relative velocity, maximizing the drag
- From the concentration profiles, fibres show more gravitational settling than spheres of the same size
- Streamwise rms velocity of fibres is more asymmetric than that of spheres
- Vertical rms velocity is remarkably lower in the case of elongated cylinders than for spherical particles, as happens with other type of irregular shapes