

***3D-CFD-Simulations of the gas-particle
flow in a cold gas Laval nozzle to
predict the mechanical erosion***

***14th Workshop on Two-Phase Flow
Predictions
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- **Introduction**

- 3D CFD of multiphase flows in power plants at CHP Cottbus as an example
- Motivation
- State of the art
- Dominant wear mechanisms

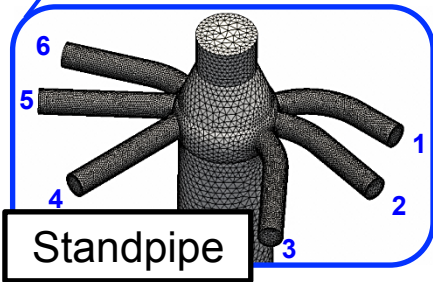
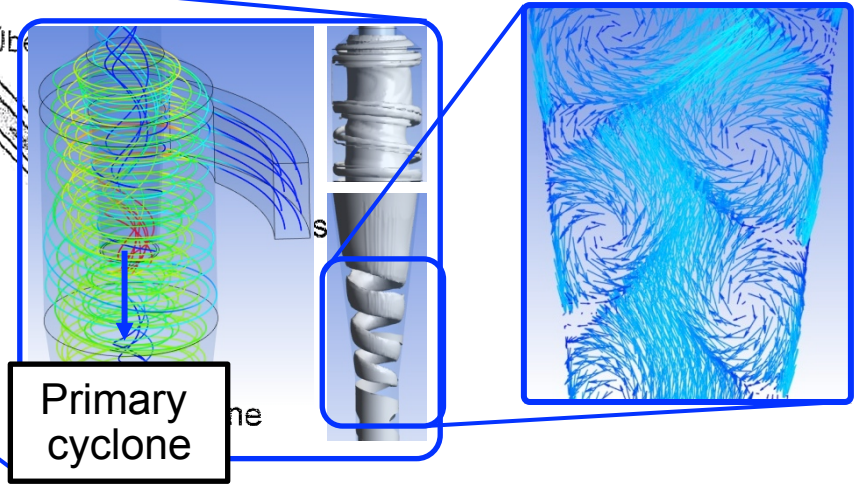
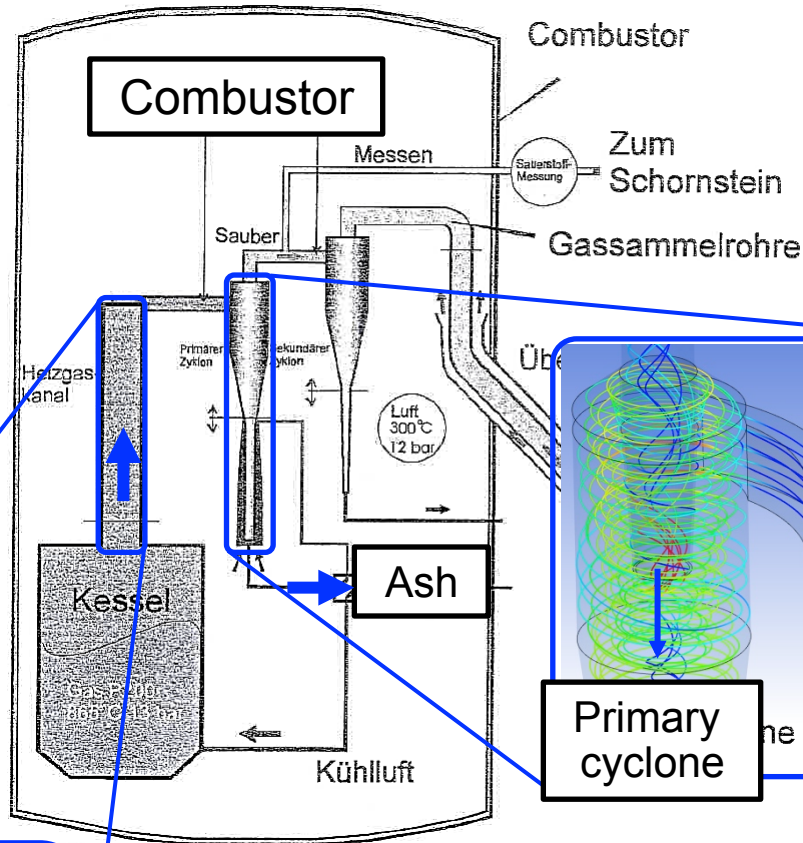
- **Numerical methods**

- **Numerical set-up**

- **Results**

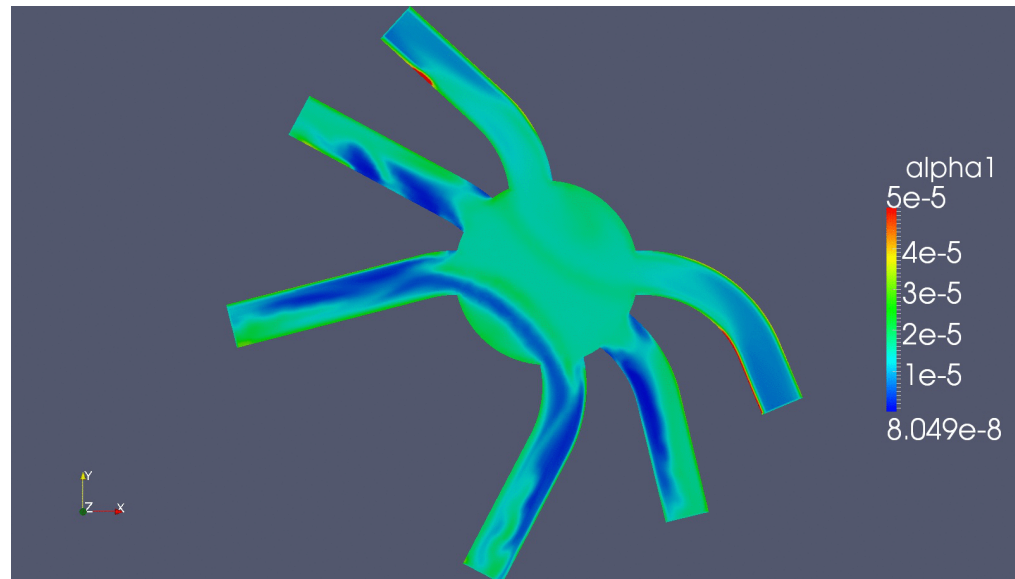
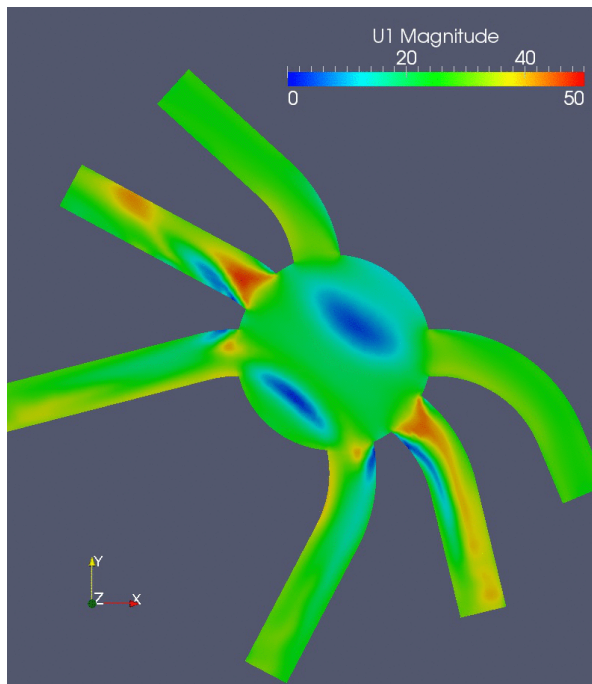
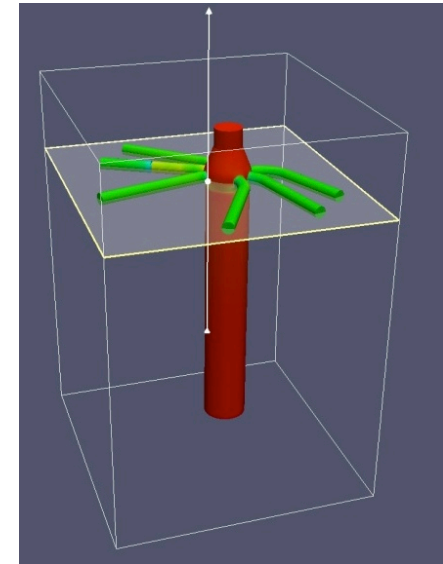
- **Outlook**

3D CFD of multiphase flows in power plants at CHP Cottbus as an example



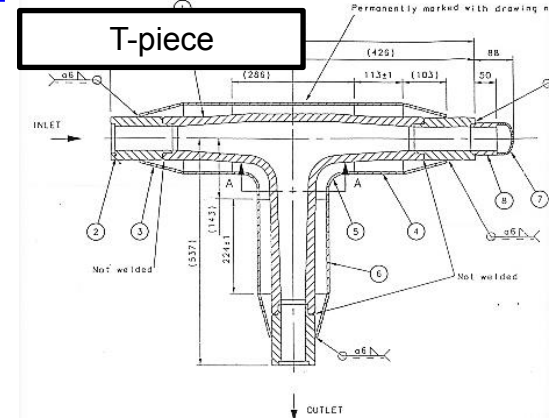
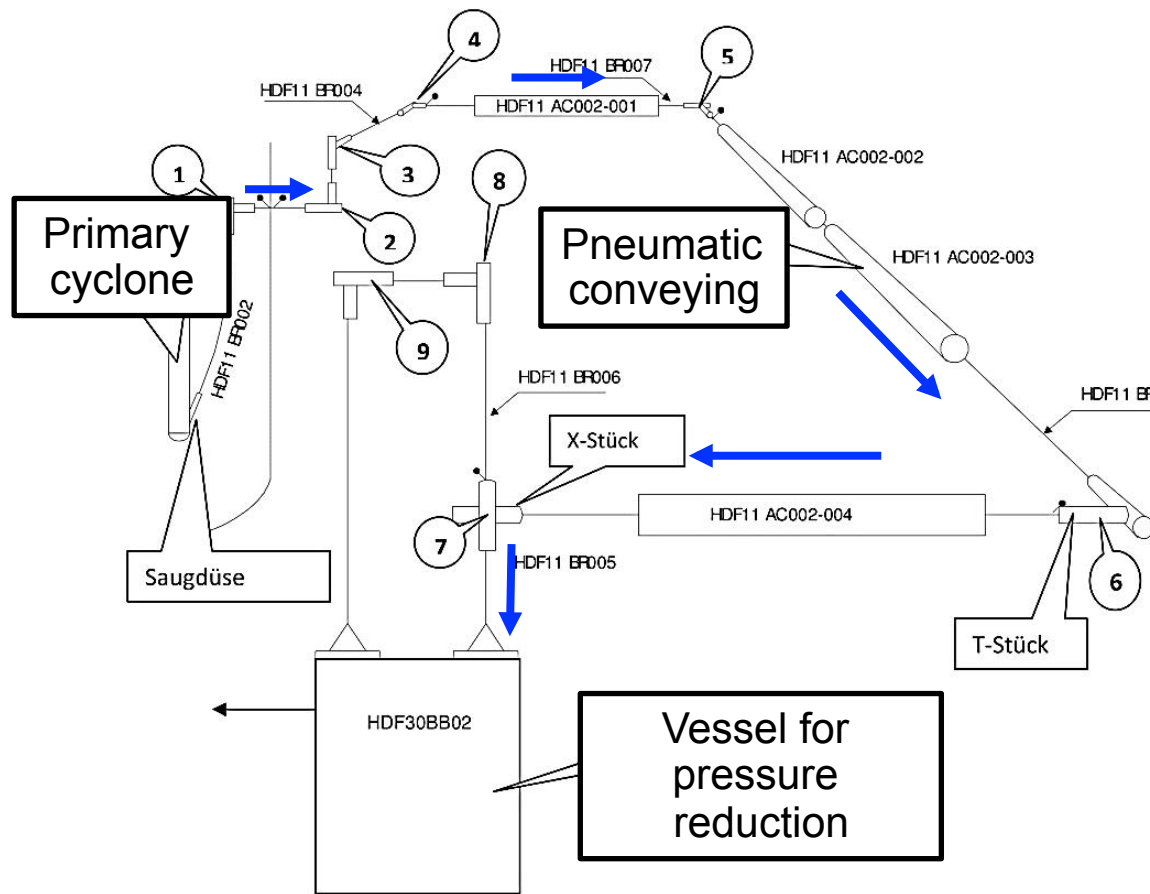
3D CFD of multiphase flows in power plants at CHP Cottbus as an example

- Prediction of the two-phase flow in the standpipe
- Non homogeneous mass fluxes at the outlets
- Euler/Euler method



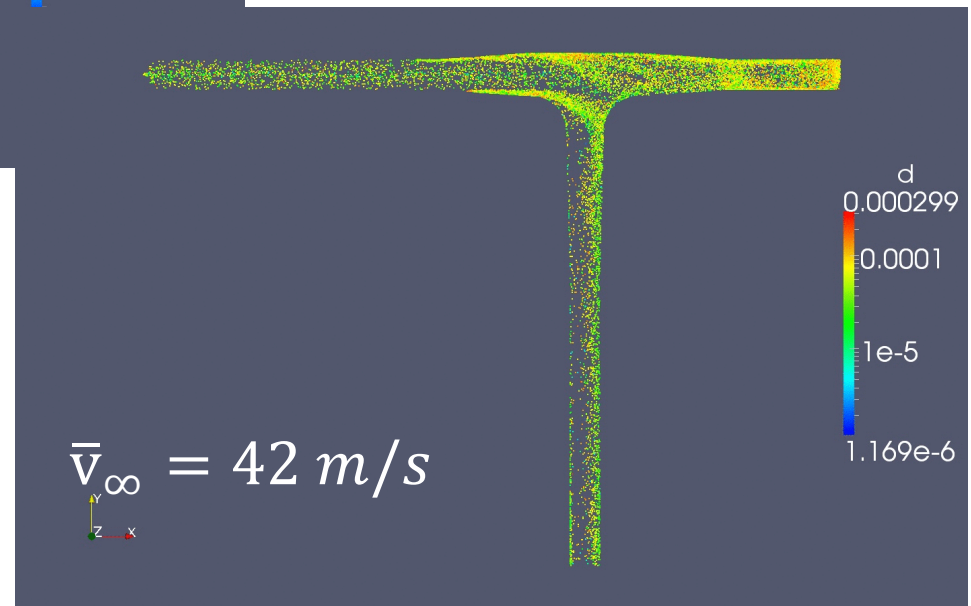
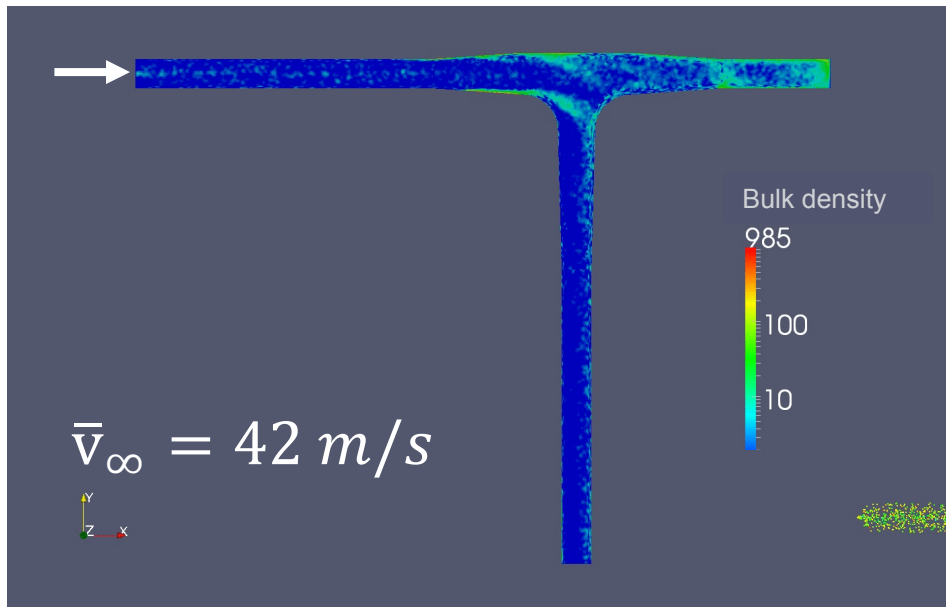
3D CFD of multiphase flows in power plants at CHP Cottbus as an example

Overview of the pneumatic ash transport system:



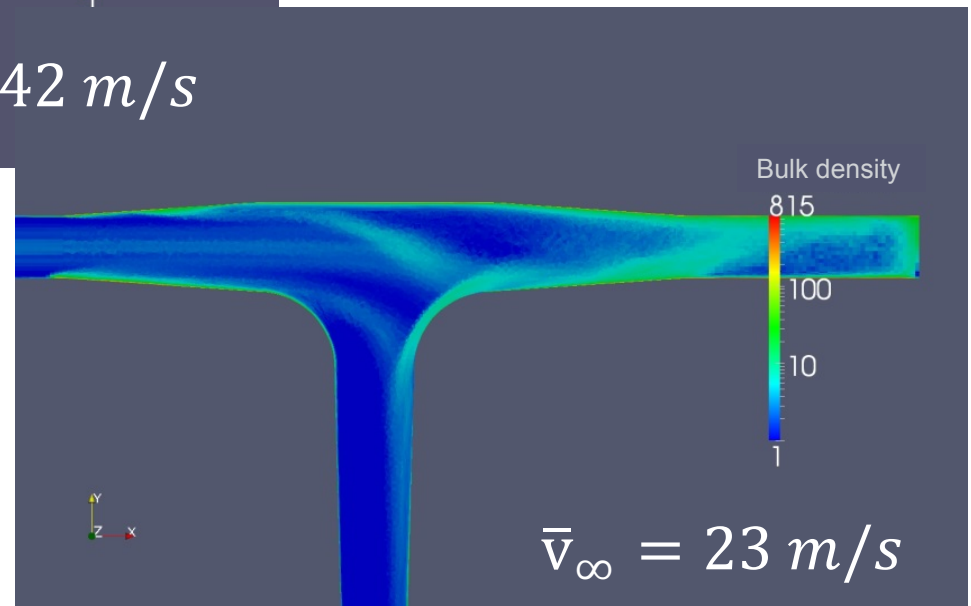
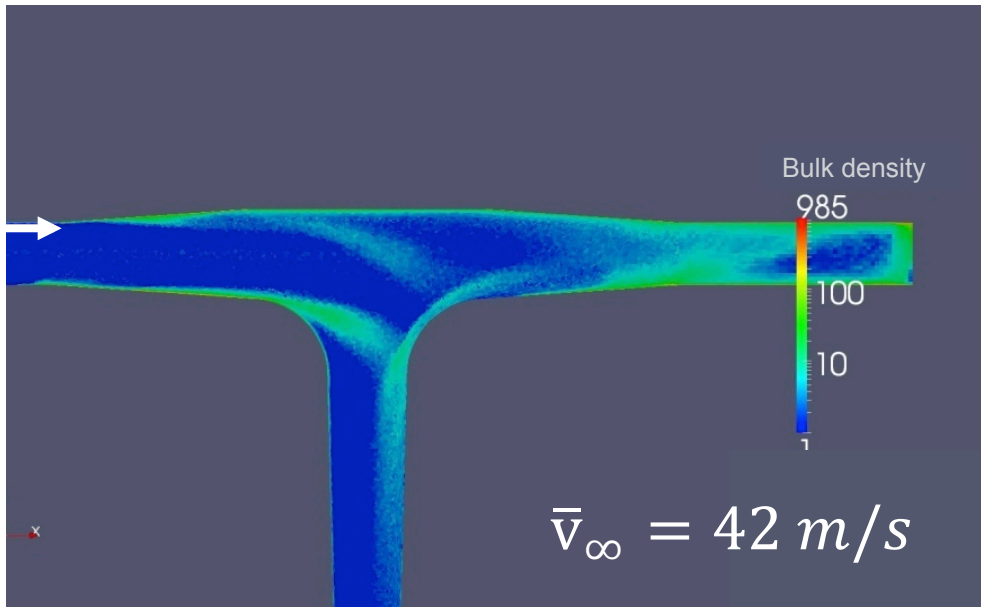
Numerical simulations in cooperation with the Cottbus power plant

Euler/Lagrange simulation at a T-piece



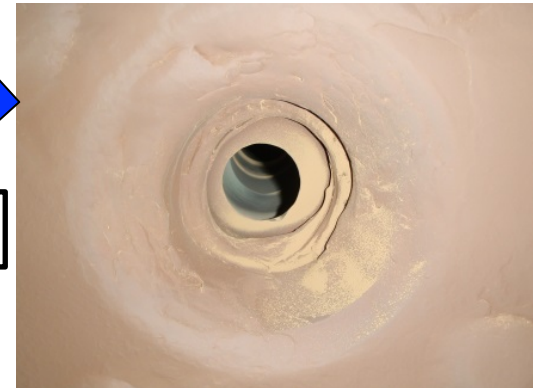
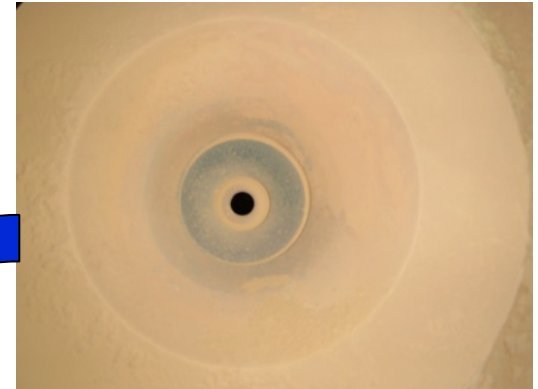
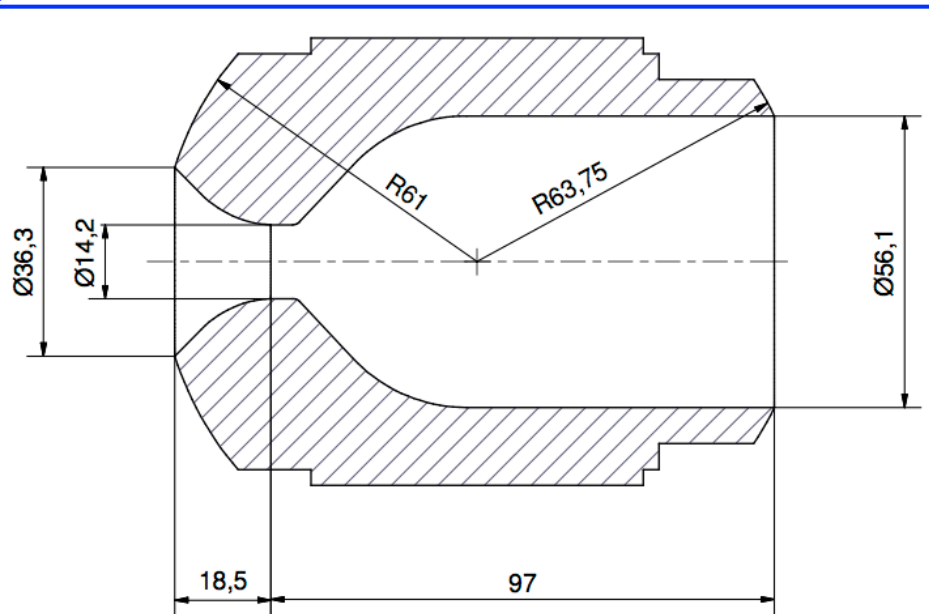
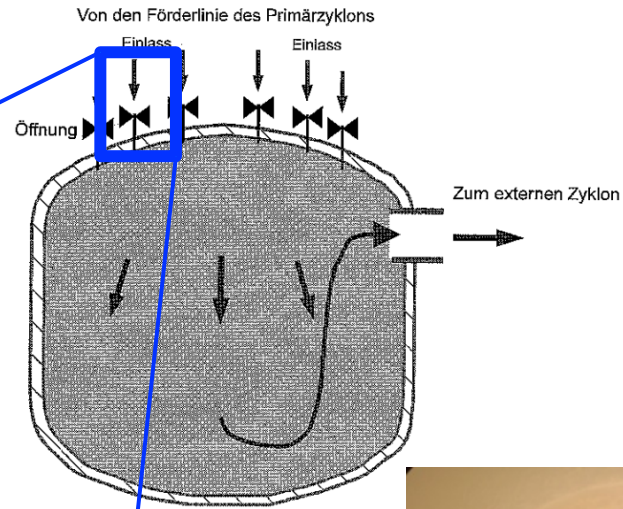
Numerical simulations in cooperation with the Cottbus power plant

Time averaged:



Motivation

Vessel for pressure reduction:



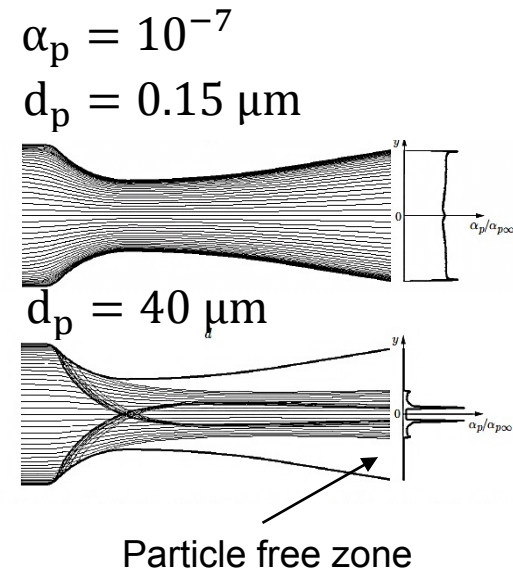
Erosion

Nomenclature

Added mass force	\vec{F}_{AM}	Volume fraction of the particle phase	$\alpha_p = \frac{V_p}{V}$
Density	ρ	Bulk density	$\rho_b = \alpha_p \cdot \rho_p$
Diameter	d	Mach number	$Ma = \frac{ \nabla }{c}$
Diffusive flux of heat	\vec{j}	Mass load	$\mu = \frac{m_p}{m_c}$
Drag Force	\vec{F}_D	Particle response time	$\tau_p = \frac{\rho_p d_p^2}{18\mu f_D}$
Dynamic viscosity	μ	Reynolds number	$Re = \frac{ \nabla \cdot d \cdot \rho_c}{\mu}$
Mass	m	Stokes number	$St = \frac{\tau_p \nabla }{d}$
Momentum source due to particle forces	\vec{S}	Subscript	
Pressure	p	Inlet condition	∞
Pressure gradient force	\vec{F}_{PG}	Particle phase	p
Speed of sound	c	Continuous phase	c
Temperature	T		
Volume	V		
Velocity	\vec{v}		
Viscous stress Tensor	T		
Work due to particle forces	W		

Particle tracking in a Laval nozzle

- Exp: Lu et al. 2011
 - showed that the flow through a Laval nozzle is less sensitive to the gas flow rate and pressure disturbances
 - made it easier to control coal mass flow rate precisely
- Num: Verevkin, Tsirkunov. 2008
 - Increasing Stokes number → higher particle inertia
 - Separation from streamlines → Particle free zone
 - Number of colliding particles increases
 - Higher momentum of impacting particles
 - Large particle concentrates around centerline

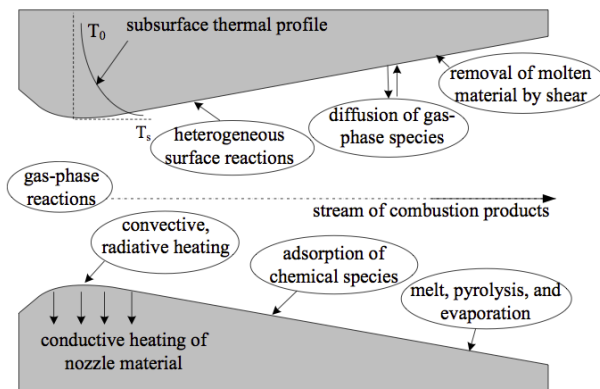


Lu et al. 2011. *Flow characteristics and pressure drop across the Laval nozzle in dense phase pneumatic conveying of the pulverized coal*. Chemical Engineering and Processing, 50

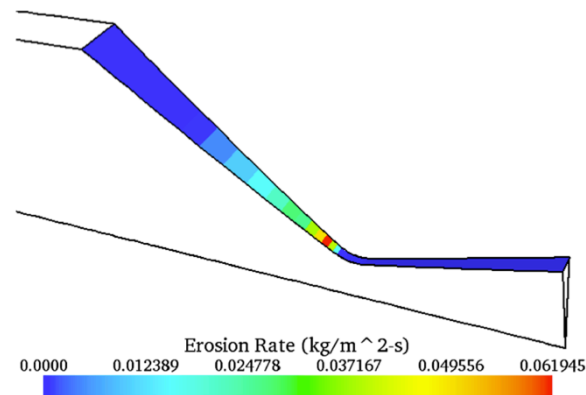
Verevkin, Tsirkunov. 2008. *Flow of a dispersed phase in the Laval nozzle and in the test section of a two-phase hypersonic shock tunnel*. Journal of Applied Mechanics and Technical Physics, 49

Erosion due to particles impacts in a Laval nozzle

- Num: Thakre. 2008
 - Mechanical erosion caused by the impingement of particles is negligible in the throat region, it is not significant to total mass loss
- Num: Thakre et al. 2013
 - Mechanical erosion is prevalent only in the convergent section of the rocket nozzle



$$d_p = 50 \dots 80 \mu\text{m}, \mu = 0.006 \dots 0.08$$

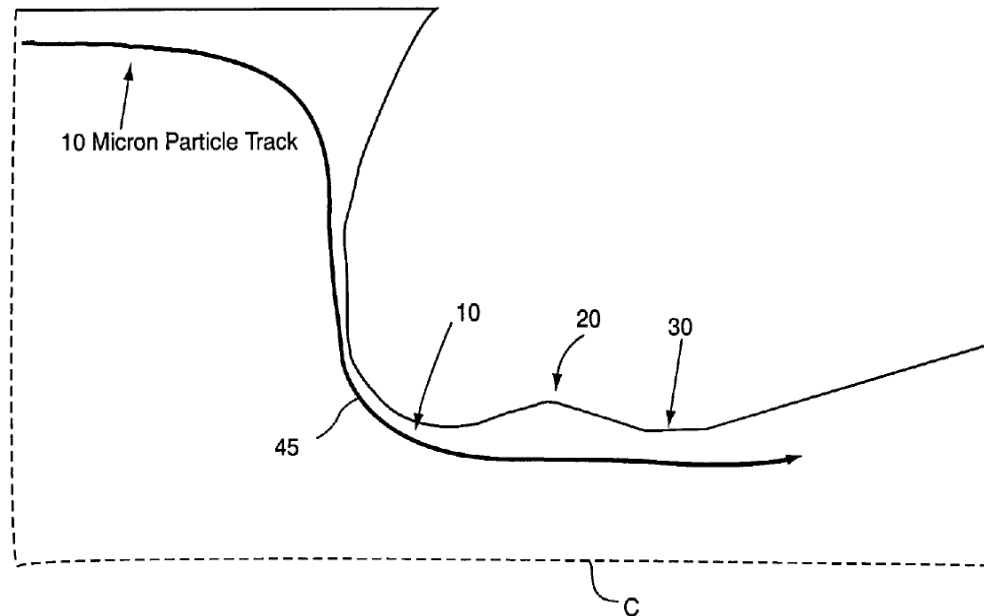


Thakre. 2008. *Chemical erosion of graphite and refractory metal nozzles and its mitigation in solid-propellant rocket motor*. Thesis

Thakre et al. 2013. *Mechanical erosion of graphite nozzle in solid-propellant rocket motor*. Journal of Propulsion and Power, 29

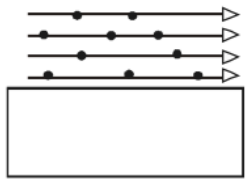
Optimized shape

- Num: Hennessey. 2001

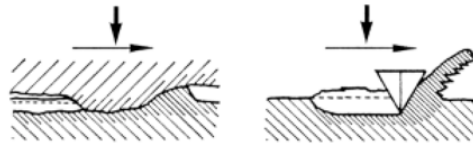


- ➔ Flow predictions of Laval nozzles are usually performed for rocket nozzles
- Different conditions like temperatures up to 3000 K
 - Reacting and/or melting particles
 - Main erosion in these cases is due to chemical erosion and melting surfaces whereas mechanical erosion is neglected
 - Furthermore the lifetime of such nozzles is in the order of minutes compared to thousands of hours in our case.

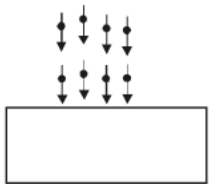
Dominant wear mechanisms



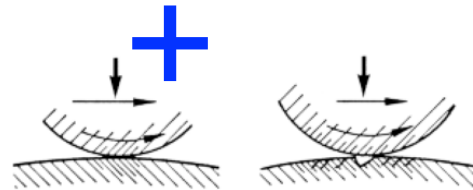
Streaming



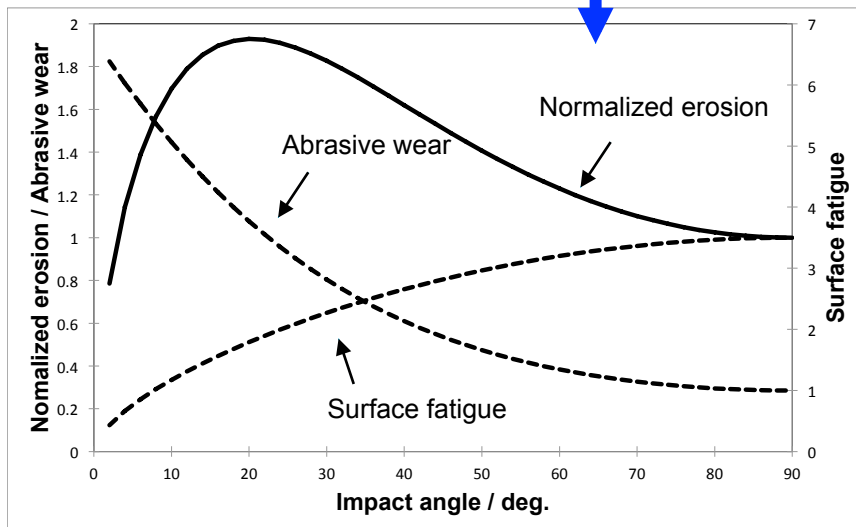
Abrasive wear



Bouncing

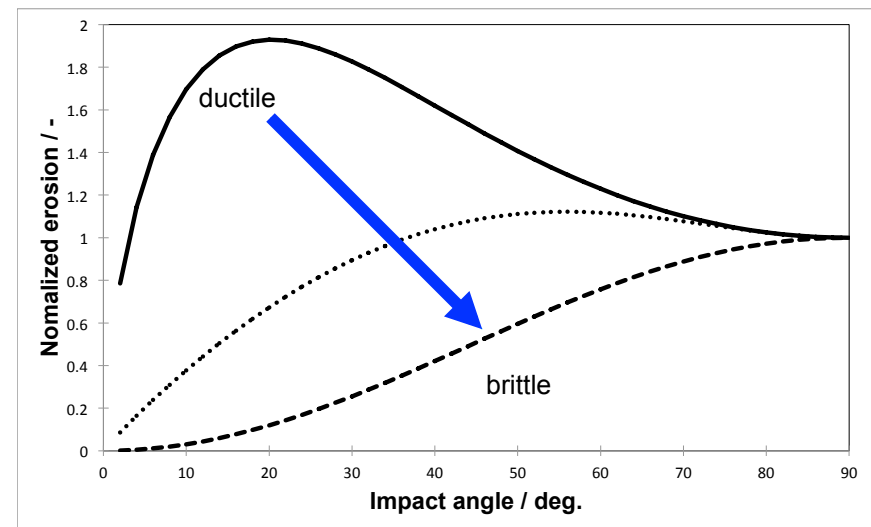


Surface fatigue



Depends on material:

- ductile cutting and plastic deformation
- brittle fracture



Numerical methods

- Open source software package OpenFOAM
- Density based compressible solver rhoCentralFoam
 - does not involve Riemann solver
 - based on central-upwind schemes

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{u}) = 0$$

$$\frac{\partial(\rho \vec{u})}{\partial t} + \nabla \cdot [\vec{u}(\rho \vec{u})^T] + \nabla p + \nabla \cdot T = \vec{S}$$

≠ 0 for 2/4-way-coupling

$$\frac{\partial(\rho E)}{\partial t} + \nabla \cdot [\vec{u}(\rho E)] + \nabla \cdot (p \vec{u}) + \nabla \cdot (T \cdot \vec{u}) + \nabla \cdot \vec{j} = W$$

Weller et al. 1998. *A tensorial approach to computational continuum mechanics using object-oriented techniques*. Computers in Physics, 12

Greenshields et al. 2009. *Implementation of semi-discrete, non-staggered central schemes in a colocated, polyhedral, finite volume framework, for high-speed viscous flows*. Int. J. Numer. Meth. Fluids

Marcantoni et al. 2012. *High speed flow simulations using OpenFOAM*. Mecánica Computacional, 2012

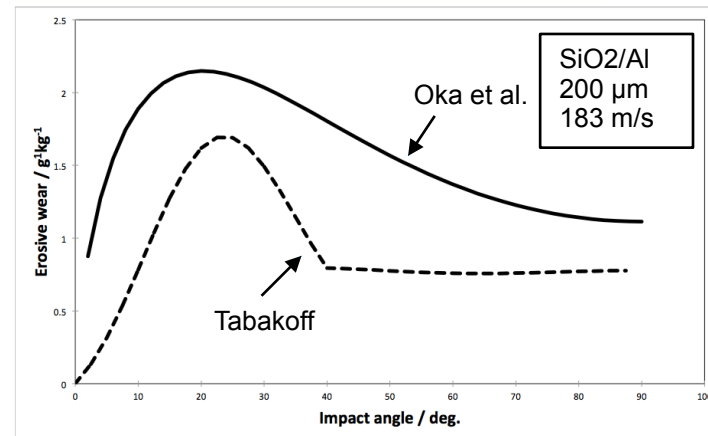
- Combination of the compressible solver with particle methods supplied by OpenFOAM → Euler/Lagrangian coupling

- Equation of motion

$$m_p \frac{d\vec{v}_p}{dt} = \vec{F}_D + \vec{F}_{PG} + \vec{F}_{AM}$$

- Drag force by Henderson, takes high Mach number into account
- Stochastic Particle-Wall collisions

- Erosion modeling by Tabakoff



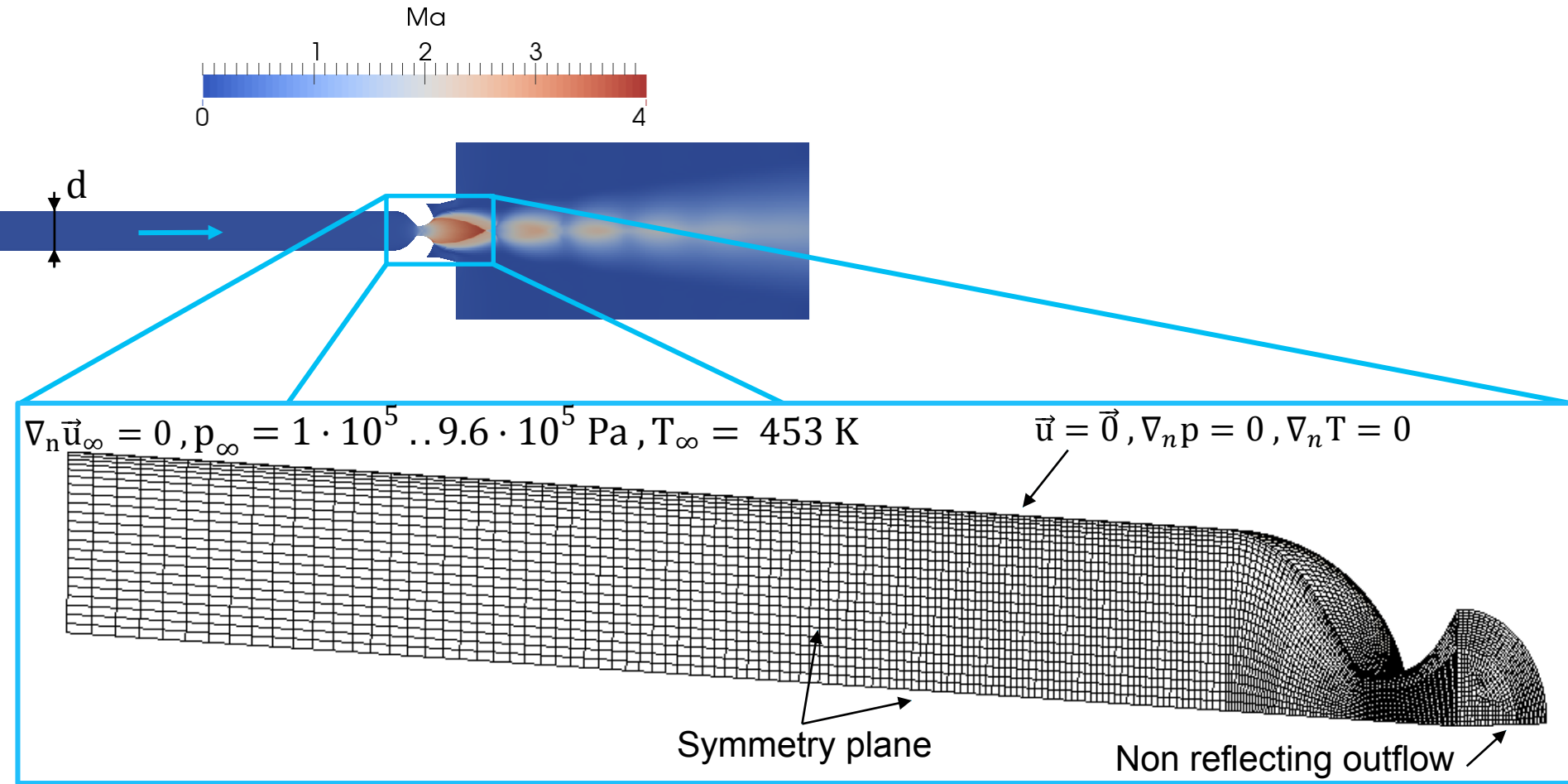
Sommerfeld, Huber. 1999. *Experimental analysis and modelling of particle-wall collision*. International Journal of Multiphase Flow, 25

Henderson. 1976. Drag coefficients of spheres in continuum and rarefied flows. AIAA Journal, 14

Grant, Tabakoff. 1975. *Erosion prediction in turbomachinery resulting from environmental solid particles*. J. Aircraft, 12

Oka et al. 2005. *Practical estimation of erosion damage caused by solid particle impact part 1&2*. Wear, 259

Numerical set-up

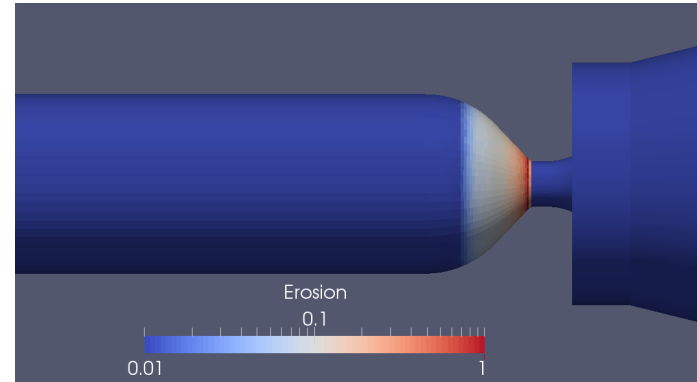
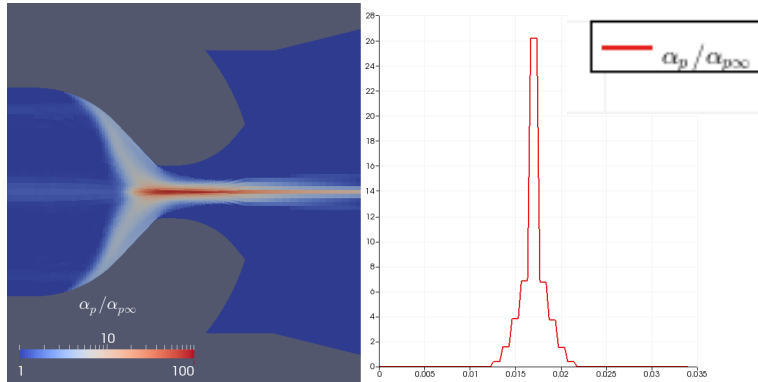


- 72716 cells
- Time step according to $\text{CFL} < 0.25$
- SST $k-\omega$ turbulence model
- $\text{Re}_\infty \sim 300,000$

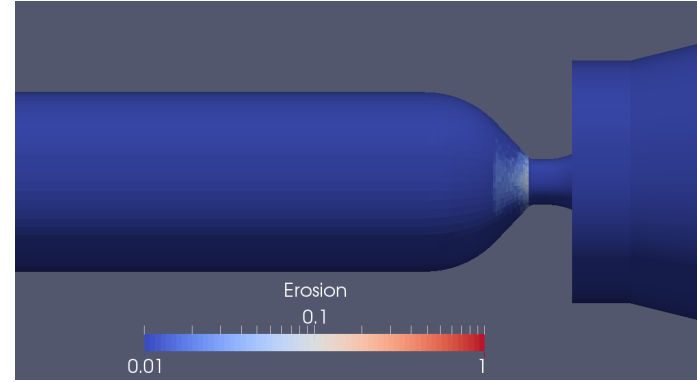
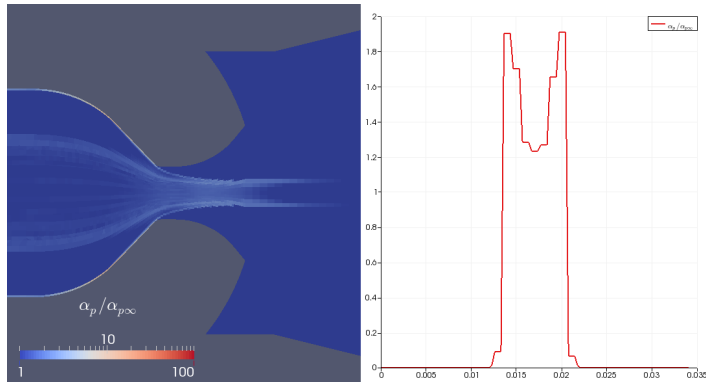
- $\mu = 0.3$
- $d_p = 1, 10, 100 \mu\text{m}$
- $\rho_p = 2500 \text{ kg/m}^3$
- $1e7$ parcels per second, 220,000 at once
- 1 way coupling

Results/Qualitative comparison

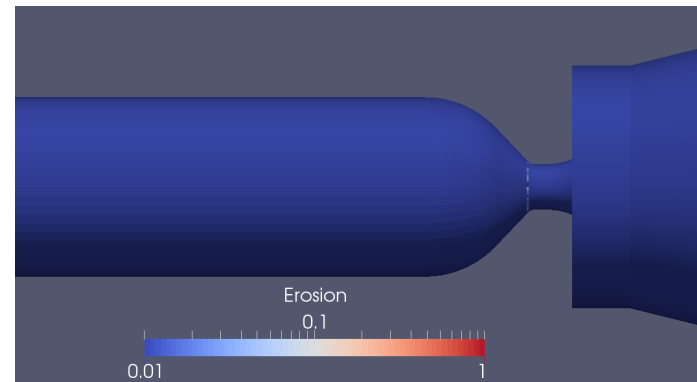
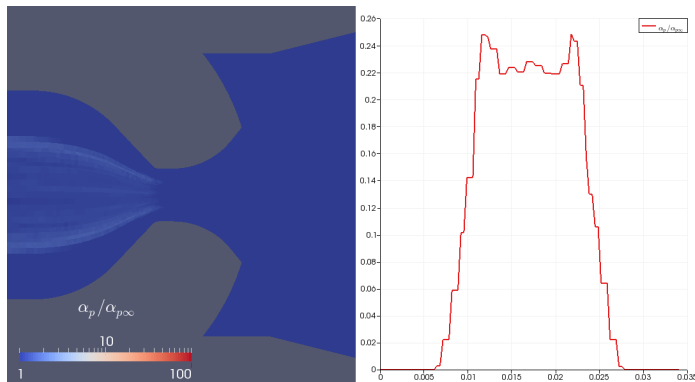
100 μm ,
St = 97



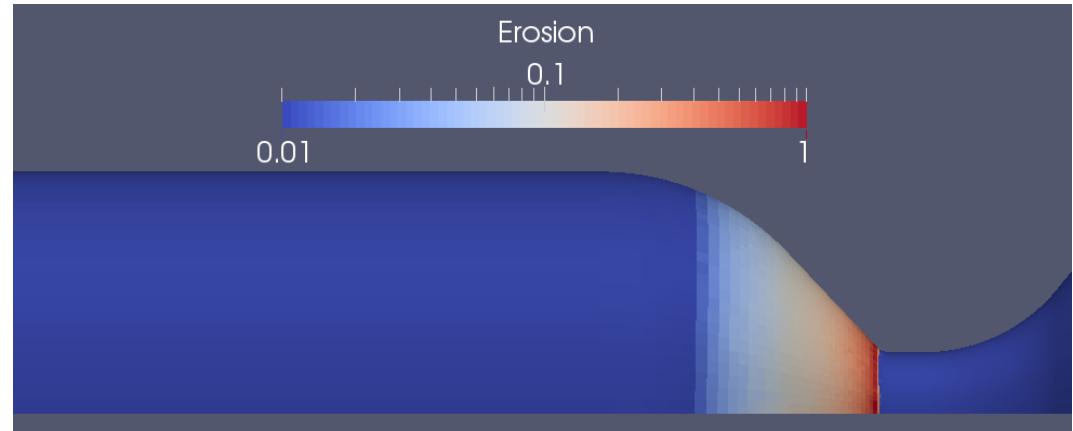
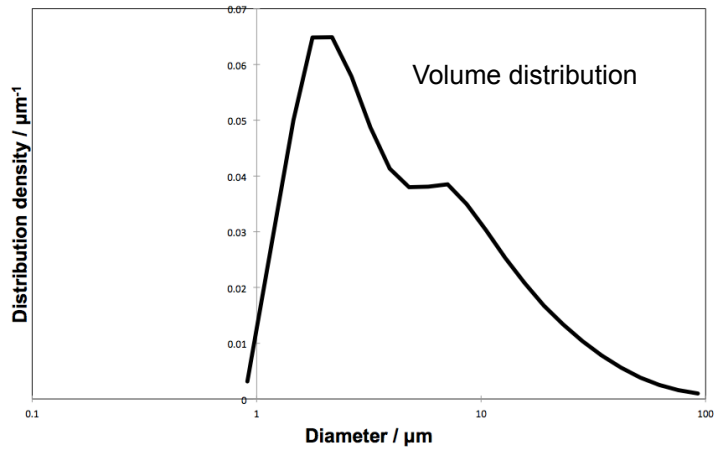
10 μm ,
St = 3



1 μm ,
St = 0.05



Results/Qualitative comparison



- It could be observed that particles with $St > 1$ strongly tended to collide with the nozzle wall
- Erosion mainly due to the largest particles
- Erosion located in a small area in front of the throat
- Qualitatively comparable with numerical and experimental research of two phase flow Laval nozzles

- Two-way and four-way coupling is necessary to achieve more accurate results
- particle-particle collisions will be modeled by the stochastic method of O'Rourke
- Modification of the Navier-Stokes equation which takes dispersed phase volume fraction into account
- Verification:
 - Quantitative comparison with pressure drop according to Lu et al. 2011
 - Comparison of the predicted erosion with Thakre et al. 2013
- Variation of geometry, material and mass load

Thank you for your attention.